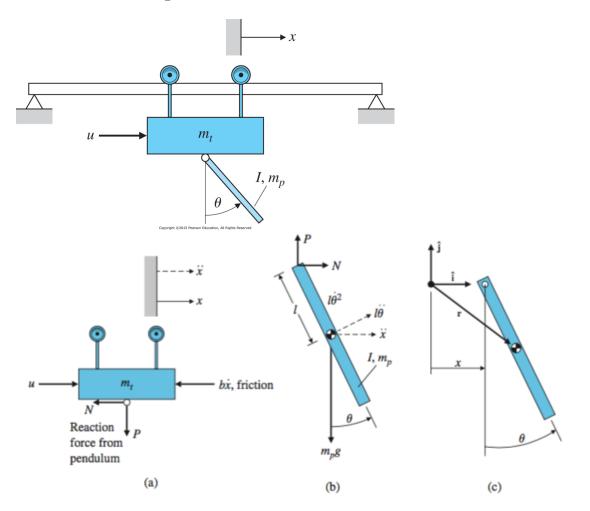
(Revisited Example 2.8)



For the pendulum,

$$N = m_p \ddot{x} + m_p l \ddot{\theta} \cos \theta - m_p l \dot{\theta}^2 \sin \theta$$
$$P - m_p g = m_p l \ddot{\theta} \sin \theta + m_p l \dot{\theta}^2 \cos \theta$$
$$-Pl \sin \theta - Nl \cos \theta = I \ddot{\theta} \quad \rightarrow \quad (I + m_p l^2) \ddot{\theta} + m_p g l \sin \theta = -m_p \ddot{x} l \cos \theta$$

••

For the crane,

$$m_t \ddot{x} = u - N - b\dot{x} \quad \rightarrow \quad (m_t + m_p)\ddot{x} + b\dot{x} + m_p l\ddot{\theta}\cos\theta - m_p l\dot{\theta}^2\sin\theta = u$$

As a result, we have the complete equation of motion:

$$(I + m_p l^2)\ddot{\theta} + m_p g l \sin \theta + m_p \ddot{x} l \cos \theta = 0$$
$$(m_t + m_p)\ddot{x} + b\dot{x} + m_p l\ddot{\theta} \cos \theta - m_p l\dot{\theta}^2 \sin \theta = u$$

For the linearization with small angle variation, $\sin \theta \approx \theta$, $\cos \theta \approx 1$ and $\dot{\theta}^2 \approx 0$, we have

$$(I + m_p l^2)\ddot{\theta} + m_p g l\theta + m_p l\ddot{x} = 0$$
$$(m_t + m_p)\ddot{x} + b\dot{x} + m_p l\ddot{\theta} = u$$

Furthermore, ignoring the damping b, we can get the TF as follow:

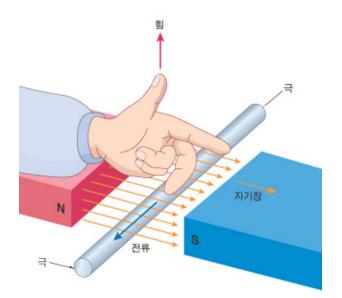
$$(I + m_p l^2)s^2\Theta(s) + m_p g l\Theta(s) + m_p ls^2 X(s) = 0$$

$$(m_t + m_p)s^2 X(s) + m_p ls^2\Theta(s) = u$$

Thus, we have

$$\therefore \qquad \frac{\Theta(s)}{U(s)} = \frac{-m_p l}{[(m_t + m_p)(I + m_p l^2) - m_p^2 l^2]s^2 + (m_t + m_p)m_p g l}$$

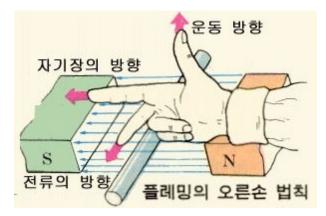
3 Models of Electromechanical Systems



- 1. Law of Motor
 - If a current i[A] in a conductor of length l[m] is arranged at right angles in a magnetic field of B[Tesla], then there is a force on the conductor at right angles to the plane of i and B with magnitude

$$F = Bil [N]$$

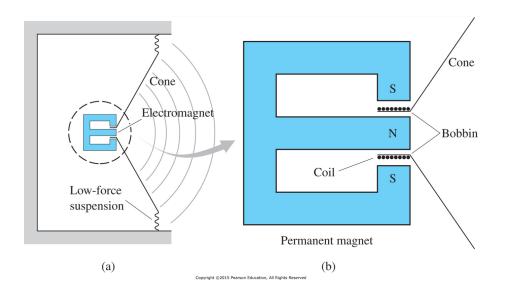
• It is called "law of motors" regarding the conversion of electric energy into mechanical work.



- 2. Law of Generator
 - If a conductor of length l[m] is moving in a magnetic field B[T] at a velocity of v[m/s] at mutually right angles, the electric voltage is established across the conductor with magnitude

e = Blv[V]

• It is called "law of generators"



3. Loudspeakers

(Example 2.13, Loudspeaker) A typical geometry for a loudspeaker for producing sound is sketched. The permanent magnet establishes a raidal field in the cylindrical gap b/w the poles of the magnet. The force on the conductor wound on the bobbin causes the voice coil to move, producing sound. The cone has mass M and viscous friction b. Assume the magnet establishes a uniform field B of 0.4[T] and the bobbin has 18 turns at a 1.9-cm diameter.

• The conductor length is

$$l = (2 \cdot \pi \cdot 0.0095) \cdot 18 = 1.074[m]$$

• The force is

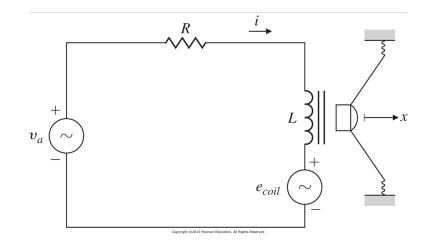
$$F = Bil = 0.4 \cdot i \cdot 1.074 = 0.43i[N]$$

• Mechanical system is modeled as

$$M\ddot{x} = F - b\dot{x} \rightarrow M\ddot{x} + b\dot{x} = 0.43i$$

• The TF of mechanical part is

$$\frac{X(s)}{I(s)} = \frac{0.43}{s(Ms+b)}$$



(Example 2.14, Loudspeaker with Circuit) Consider the driving circuit for the louspeaker. Find the differential equation relating the input voltage v_a and the output cone displacement x. Assume the effective resistance R and inductance L.

• The resulting voltage according to the speaker motion is

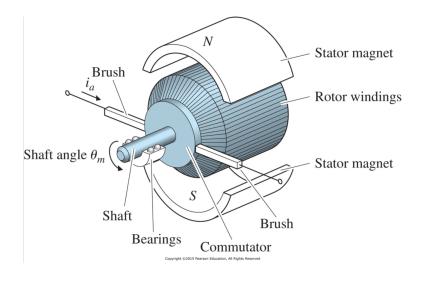
$$e_{coil} = Bl\dot{x} = 0.4 \cdot 1.047 \cdot \dot{x} = 0.43\dot{x}$$

• Due to the induced voltage effect, the electric circuit is modeled as

$$L\frac{di}{dt} + Ri + 0.43\dot{x} = v_a$$
$$(Ls + R)I(s) + 0.43sX(s) = V_a(s)$$
$$(Ls + R)\left(\frac{s(Ms + b)}{0.43}\right)X(s) + 0.43sX(s) = V_a(s)$$

• The TF of dynamic model for the loudspeaker is

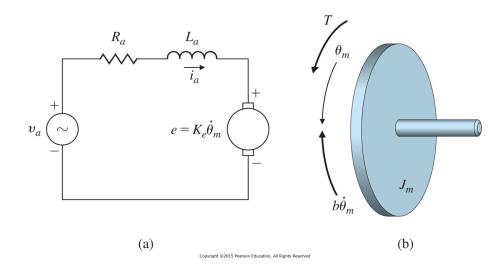
$$\therefore \qquad \frac{X(s)}{V_a(s)} = \frac{0.43}{s[(Ms+b)(Ls+R)+0.43^2]}$$



- 4. Motors
 - Consider DC motor. The motor equations give the torque T on the rotor in terms of armature current i_a and express the back emf voltage in terms of shaft's rotational velocity $\dot{\theta}_m$.

$$T(t) = r \times (B \cdot i_a(t) \cdot l) = (rBl) \cdot i_a(t) = K_t \cdot i_a(t)$$
$$e(t) = B \cdot l \cdot (r \times \dot{\theta}_m(t)) = (rBl) \cdot \dot{\theta}_m(t) = K_e \cdot \dot{\theta}_m(t)$$

where r implies an effective moment arm of motor, K_t and K_e denote the motor torque constant and the motor back emf constant. Note that $K_t = K_e$ with different dimensions.



(Example 2.15, Modeling a DC Motor) Assume the rotor has inertia J_m and viscous friction coefficient b.

$$J_m \dot{\theta}_m + b \dot{\theta}_m = K_t i_a$$
$$L_a \frac{di_a}{dt} + R_a i_a + K_e \dot{\theta}_m = v_a$$

• Take LT with zero initial conditions, then we have

$$(J_m s^2 + bs)\Theta_m(s) = K_t I_a(s)$$
$$(L_a s + R_a)I_a + K_e s\Theta_m(s) = V_a(s)$$

• Also, we can get the TF as follow:

$$\cdot \quad \frac{\Theta_m(s)}{V_a(s)} = \frac{K_t}{s[(J_m s + b)(L_a s + R_a) + K_t K_e]}$$

• Ignoring the inductance due to small quantity, we can simplify above complete model into

$$\therefore \quad \frac{\Theta_m(s)}{V_a(s)} \approx \frac{K_t}{s[J_m R_a s + (bR_a + K_t K_e)]} \\ \approx \frac{K}{s(\tau s + 1)}$$

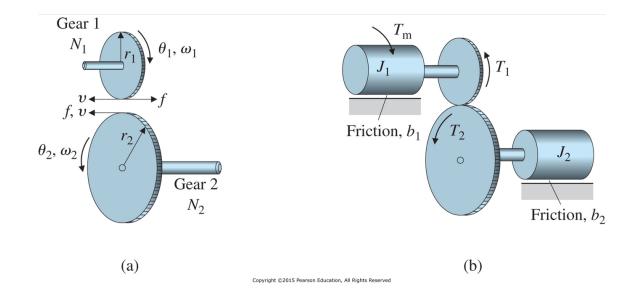
where

$$K = \frac{K_t}{bR_a + K_t K_e} \qquad \qquad \tau = \frac{J_m R_a}{bR_a + K_t K_e}$$

• If we consider the output speed, $\omega_m(t) = \dot{\theta}_m(t)$, then

$$\therefore \qquad \frac{\Omega_m(s)}{V_a(s)} \approx \frac{K}{\tau s + 1}$$

• Other types of motors: AC motor, brushless DC motor, stepping motor and so on.



- 5. Gears
 - Consider gear transmission. Since the transmitted force and velocity at the contact point are the same, we have

$$rac{T_1}{r_1} = rac{T_2}{r_2} = f$$
: force applied by teeth at the contact point
 $\omega_1 r_1 = \omega_2 r_2 = v$: velocity at the contact point
 $rac{2\pi r_1}{N_1} = rac{2\pi r_2}{N_2} = m$: module for the transmission

• Let us define the gear ratio $n = \frac{N_2}{N_1}$, then we have

$$n = \frac{N_2}{N_1} = \frac{r_2}{r_1} = \frac{\omega_1}{\omega_2} = \frac{T_2}{T_1}$$

• The equations of motion for bodies 1 and 2 are

$$J_{1}\ddot{\theta}_{1} + b_{1}\dot{\theta}_{1} = T_{m} - T_{1} \longrightarrow nJ_{1}\ddot{\theta}_{2} + nb_{1}\dot{\theta}_{2} = T_{m} - \frac{T_{2}}{n}$$

$$J_{2}\ddot{\theta}_{2} + b_{2}\dot{\theta}_{2} = T_{2} \longrightarrow J_{2}\ddot{\theta}_{2} + b_{2}\dot{\theta}_{2} = nT_{m} - n^{2}J_{1}\ddot{\theta}_{2} - n^{2}b_{1}\dot{\theta}_{2}$$

$$(J_{2} + n^{2}J_{1})\ddot{\theta}_{2} + (b_{2} + n^{2}b_{1})\dot{\theta}_{2} = nT_{m}$$

where T_m is a servo motor torque, T_1 is the reaction torque from gear 2 acting back on gear 1, and T_2 is the torque applied on gear 2 by gear 1.

• Take LT with zero initial conditions

$$\frac{\Theta_2(s)}{T_m(s)} = \frac{n}{J_{eq}s^2 + b_{eq}s}$$

where

$$J_{eq} = J_2 + n^2 J_1 \qquad \qquad b_{eq} = b_2 + n^2 b_1$$

are referred to as the equivalent inertias and damping coefficients.