

3.3 Exponential Coordinate Representation of Rigid-Body Motions

Exponential Coordinates of Rigid-Body Motions

- Chasles-Mozzi theorem: every rigid-body displacement can be expressed as a displacement along a fixed screw axis \mathcal{S} in space.
- (In the previous lecture) The exponential coordinates $\hat{\omega}\theta$ for rotation:

$$\text{matrix exponential : } \quad [\hat{\omega}]\theta \in so(3) \quad \rightarrow \quad R = e^{[\hat{\omega}]\theta} \in SO(3)$$

$$\text{matrix logarithm : } \quad R = e^{[\hat{\omega}]\theta} \in SO(3) \quad \rightarrow \quad [\hat{\omega}]\theta \in so(3)$$

- By analogy to the exponential coordinates $\hat{\omega}\theta$ for rotations R , let us define the six-dimensional exponential coordinates of a homogeneous transformation T as $\mathcal{S}\theta \in \mathfrak{R}^6$, where θ is the distance that must be traveled along the screw axis to take a frame from the origin I to T .
 - If the pitch of the screw axis $\mathcal{S} = (\omega, v)$ is finite then $\|\omega\| = 1$ and θ corresponds to the angle of rotation about the screw axis.
 - If the pitch of the screw is infinite then $\omega = 0$ and $\|v\| = 1$ and θ corresponds to the linear distance traveled along the screw axis.
- Also by analogy to the rotation case, let us define a matrix exponential (exp) and matrix logarithm (log):

$$\text{matrix exponential : } \quad [\mathcal{S}]\theta \in se(3) \quad \rightarrow \quad T = e^{[\mathcal{S}]\theta} \in SE(3)$$

$$\text{matrix logarithm : } \quad T = e^{[\mathcal{S}]\theta} \in SE(3) \quad \rightarrow \quad [\mathcal{S}]\theta \in se(3)$$

- Expanding the matrix exponential in series form leads to

$$\begin{aligned}
e^{[\mathcal{S}]\theta} &= I_{4 \times 4} + [\mathcal{S}]\theta + [\mathcal{S}]^2 \frac{\theta^2}{2!} + [\mathcal{S}]^3 \frac{\theta^3}{3!} + \dots = \begin{bmatrix} I_{3 \times 3} + [\omega]\theta + [\omega]^2 \frac{\theta^2}{2!} + [\omega]^3 \frac{\theta^3}{3!} + \dots & v\theta + [\omega]v \frac{\theta^2}{2!} + [\omega]^2 v \frac{\theta^3}{3!} + \dots \\ 0_{3 \times 1} & 1 \end{bmatrix} \\
&= \begin{bmatrix} e^{[\omega]\theta} & G(\theta)v \\ 0_{3 \times 1} & 1 \end{bmatrix}
\end{aligned}$$

where

$$\begin{aligned}
[\mathcal{S}]^2 &= \begin{bmatrix} [\omega] & v \\ 0 & 0 \end{bmatrix} \begin{bmatrix} [\omega] & v \\ 0 & 0 \end{bmatrix} = \begin{bmatrix} [\omega]^2 & [\omega]v \\ 0 & 0 \end{bmatrix} & [\mathcal{S}]^3 &= \begin{bmatrix} [\omega]^2 & [\omega]v \\ 0 & 0 \end{bmatrix} \begin{bmatrix} [\omega] & v \\ 0 & 0 \end{bmatrix} = \begin{bmatrix} [\omega]^3 & [\omega]^2 v \\ 0 & 0 \end{bmatrix} \\
G(\theta) &= I\theta + [\omega] \frac{\theta^2}{2!} + [\omega]^2 \frac{\theta^3}{3!} + [\omega]^3 \frac{\theta^4}{4!} + [\omega]^4 \frac{\theta^5}{5!} + \dots \\
&= I\theta + [\omega] \frac{\theta^2}{2!} + [\omega]^2 \frac{\theta^3}{3!} - [\omega] \frac{\theta^4}{4!} - [\omega]^2 \frac{\theta^5}{5!} + \dots \\
&= I\theta + \left(\frac{\theta^2}{2!} - \frac{\theta^4}{4!} + \dots \right) [\omega] + \left(\frac{\theta^3}{3!} - \frac{\theta^5}{5!} + \dots \right) [\omega]^2 \\
&= I\theta + (1 - \cos \theta)[\omega] + (\theta - \sin \theta)[\omega]^2
\end{aligned}$$

Proposition 3.13. *Let $\mathcal{S} = (\omega, v)$ be a screw axis. If $\|\omega\| = 1$ or if $\omega = 0$ and $\|v\| = 1$, then, for any distance $\theta \in \mathfrak{R}$ traveled along the axis, respectively,*

$$\begin{aligned}
e^{[\mathcal{S}]\theta} &= \begin{bmatrix} e^{[\omega]\theta} & (I\theta + (1 - \cos \theta)[\omega] + (\theta - \sin \theta)[\omega]^2)v \\ 0_{3 \times 1} & 1 \end{bmatrix} & e^{[\mathcal{S}]\theta} &= \begin{bmatrix} I & v\theta \\ 0_{3 \times 1} & 1 \end{bmatrix}
\end{aligned}$$

Matrix Logarithm of Rigid-Body Motions

- On the contrary, for given an arbitrary $T = (R, p) \in SE(3)$, one can always find a screw axis $S = (\omega, v)$ and a scalar θ such that

$$e^{[S]\theta} = \begin{bmatrix} R & p \\ 0_{3 \times 1} & 1 \end{bmatrix} \in SE(3) \quad \rightarrow \quad [S]\theta = \begin{bmatrix} [\omega]\theta & v\theta \\ 0_{3 \times 1} & 0 \end{bmatrix} \in se(3)$$

where $[S]\theta$ is the matrix logarithm of $T = (R, p)$.

Algorithm 3.2. Given (R, p) written as $T \in SE(3)$, find a $\theta \in [0, \pi]$ and a screw axis $S = (\omega, v) \in \mathfrak{R}^6$ (where at least one of $\|\omega\| = 1$ and $\|v\|$ is unity) such that $e^{[S]\theta} = T$. The vector $S\theta \in \mathfrak{R}^6$ comprises the exponential coordinates for T and the matrix $[S]\theta \in se(3)$ is the matrix logarithm of T .

- If $R = I$ then set $\omega = 0$, $v = \frac{p}{\|p\|}$ and $\theta = \|p\|$
- Otherwise, use the matrix logarithm on $SO(3)$ to determine ω (written as in the $SO(3)$ algorithm) and θ for R . Then v is calculated as

$$v = G(\theta)^{-1}p$$

where

$$G(\theta)^{-1} = \frac{1}{\theta}I - \frac{1}{2}[\omega] + \left(\frac{1}{\theta} - \frac{1}{2} \cot \frac{\theta}{2} \right) [\omega]^2$$

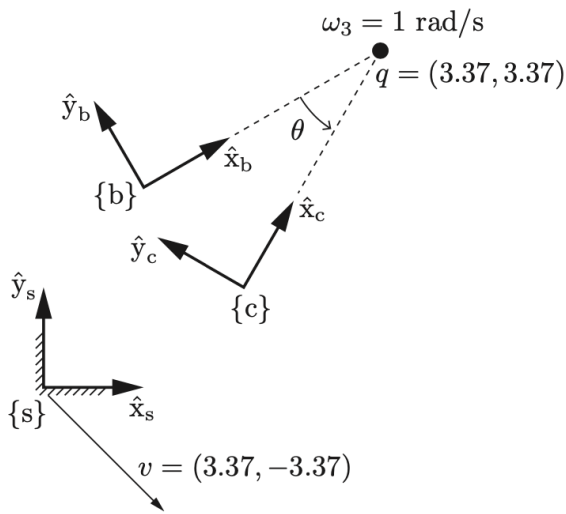


Figure 3.20: Two frames in a plane.

Example 3.4. Assume the rigid-body motion is confined to the $\hat{x}_s - \hat{y}_s$ plane. The initial frame {b} and final frame {c} can be represented by the $SE(3)$ matrices

$$T_{sb} = \begin{bmatrix} \cos 30^\circ & -\sin 30^\circ & 0 & 1 \\ \sin 30^\circ & \cos 30^\circ & 0 & 2 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$T_{sc} = \begin{bmatrix} \cos 60^\circ & -\sin 60^\circ & 0 & 2 \\ \sin 60^\circ & \cos 60^\circ & 0 & 1 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

- Assume the exponential coordinates are expressed in the fixed frame $\{s\}$, then we have

$$e^{[S]\theta} T_{sb} = T_{sc} \quad \rightarrow \quad e^{[S]\theta} = T_{sc} T_{sb}^{-1} = \begin{bmatrix} \frac{\sqrt{3}}{2} & -\frac{1}{2} & 0 & 2.134 \\ \frac{1}{2} & \frac{\sqrt{3}}{2} & 0 & -1.232 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} = T(R, p)$$

- Find θ from the rotation matrix

$$\theta = \cos^{-1} \left(\frac{\text{tr}(R) - 1}{2} \right) = \cos^{-1} \frac{\sqrt{3}}{2} = 30^\circ = 0.5236 \text{ rad}$$

- Find $\hat{\omega}$ from the rotation matrix

$$[\hat{\omega}] = \frac{1}{2 \sin \theta} (R - R^T) = \begin{bmatrix} 0 & -1 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 0 & -\hat{\omega}_3 & \hat{\omega}_2 \\ \hat{\omega}_3 & 0 & -\hat{\omega}_1 \\ -\hat{\omega}_2 & \hat{\omega}_1 & 0 \end{bmatrix} \quad \rightarrow \quad \hat{\omega} = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

- Find v from $v = G(\theta)^{-1} p$

$$\begin{aligned} v = G(\theta)^{-1} p &= \left[\frac{1}{\theta} I - \frac{1}{2} [\omega] + \left(\frac{1}{\theta} - \frac{1}{2} \cot \frac{\theta}{2} \right) [\omega]^2 \right] p \\ &= \begin{bmatrix} 1.9096 & \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} & -0.5 & \begin{bmatrix} 0 & -1 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} & + 0.0439 & \begin{bmatrix} -1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & 0 \end{bmatrix} \end{bmatrix} \begin{bmatrix} 2.134 \\ -1.232 \\ 0 \end{bmatrix} \end{aligned}$$

$$v = \begin{bmatrix} 1.8658 & 0.5 & 0 \\ -0.5 & 1.8658 & 0 \\ 0 & 0 & 1.9096 \end{bmatrix} \begin{bmatrix} 2.134 \\ -1.232 \\ 0 \end{bmatrix} = \begin{bmatrix} 3.37 \\ -3.37 \\ 0 \end{bmatrix}$$

- Find the screw axis \mathcal{S} , and the exponential coordinates (normalized twist)

$$\mathcal{S} = \begin{bmatrix} \omega \\ v \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ 3.37 \\ -3.37 \\ 0 \end{bmatrix} \quad \mathcal{S}\theta = \begin{bmatrix} \omega \\ v \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0.5236 \\ 1.7622 \\ -1.7622 \\ 0 \end{bmatrix}$$

- Alternatively,
 - we can observe that the displacement is not a pure translation - T_{sb} and T_{sc} have rotation components that differ by an angle of $\theta = 30^\circ$ and $\omega_z = 1$.
 - we can also graphically determine the point $q = (q_x, q_y)$ in the $\hat{x}_s - \hat{y}_s$ plane through which the screw axis passes; for our example this point is given by $q = (3.37, 3.37)$.

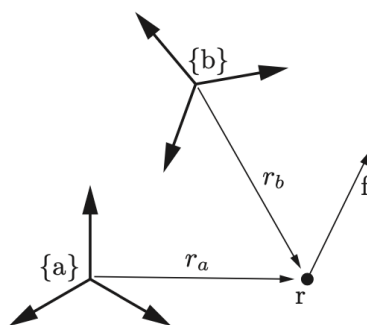


Figure 3.21: Relation between wrench representations \mathcal{F}_a and \mathcal{F}_b .

4 Wrenches

- Consider a linear force f acting on a rigid body at a point r .
- Defining a reference frame $\{a\}$, the point r can be represented as $r_a \in \mathbb{R}^3$ and the force f can be represented as $f_a \in \mathbb{R}^3$.
- This force creates a torque or moment $m_a \in \mathbb{R}^3$ in the $\{a\}$ frame:

$$m_a = r_a \times f_a$$

- Just as with twists, we can merge the moment and force into a single six-dimensional spatial force, or wrench, expressed in the $\{a\}$ frame,

$$\mathcal{F}_a = \begin{bmatrix} m_a \\ f_a \end{bmatrix} \in \mathbb{R}^6$$

- A wrench with a zero linear component is called a pure moment.

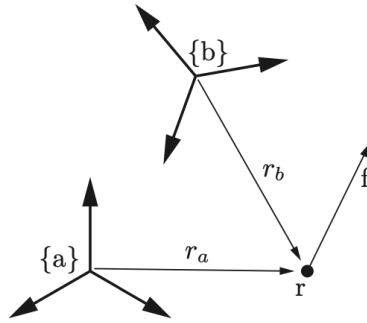


Figure 3.21: Relation between wrench representations \mathcal{F}_a and \mathcal{F}_b .

- A wrench and twist in the $\{a\}$ frame can be represented in another frame $\{b\}$
- Since the power is coordinate-independent quantity, we have

$$\mathcal{V}_b^T \mathcal{F}_b = \mathcal{V}_a^T \mathcal{F}_a = ([Ad_{T_{ab}}] \mathcal{V}_b)^T \mathcal{F}_a = \mathcal{V}_b^T [Ad_{T_{ab}}]^T \mathcal{F}_a$$

Proposition 3.14. *Given a wrench F , represented in $\{a\}$ as \mathcal{F}_a and in $\{b\}$ as \mathcal{F}_b , two representations are related by*

$$\mathcal{F}_b = [Ad_{T_{ab}}]^T \mathcal{F}_a = Ad_{T_{ab}}^T \mathcal{F}_a$$

$$\mathcal{F}_a = [Ad_{T_{ba}}]^T \mathcal{F}_b = Ad_{T_{ba}}^T \mathcal{F}_b$$

- Spatial wrench \mathcal{F}_s in $\{s\}$ and body wrench \mathcal{F}_b in $\{b\}$

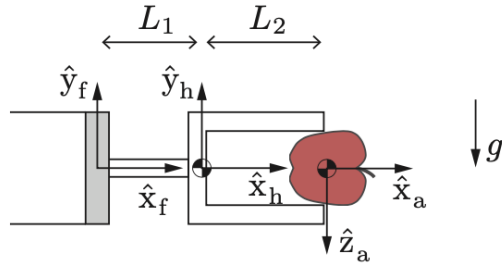


Figure 3.22: A robot hand holding an apple subject to gravity.

Example 3.5. What is the force and torque measured by the six-axis force-torque sensor b/w the hand and the arm? (mass of apple : $0.1[kg]$, mass of hand : $0.5[kg]$, $g = 10[m/s^2]$, $L_1 = 0.1[m]$, and $L_2 = 0.15[m]$) where $\{f\}$ is attached to the force-torque sensor, $\{h\}$ to CoM of hand, $\{a\}$ to CoM of apple

- Gravitational wrench on the hand in $\{h\}$ is given by

$$\mathcal{F}_h = (0, 0, 0, 0, -5N, 0)$$

- Gravitational wrench on the apple in $\{a\}$ is given by

$$\mathcal{F}_a = (0, 0, 0, 0, 0, 1N)$$

- Transformation matrices

$$T_{hf} = \begin{bmatrix} 1 & 0 & 0 & -0.1[m] \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$T_{af} = \begin{bmatrix} 1 & 0 & 0 & -0.25[m] \\ 0 & 0 & 1 & 0 \\ 0 & -1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

- Wrench measured by the six-axis force-torque sensor is

$$\begin{aligned}
\mathcal{F}_f &= [Ad_{T_{hf}}]^T \mathcal{F}_h + [Ad_{T_{af}}]^T \mathcal{F}_a \\
&= \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0.1 & 0 & 1 & 0 \\ 0 & -0.1 & 0 & 0 & 0 & 1 \end{bmatrix}^T \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ -5 \\ 0 \end{bmatrix} + \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & -1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0.25 & 0 & 0 & 0 & 1 \\ 0 & 0 & -0.25 & 0 & -1 & 0 \end{bmatrix}^T \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ -0.5 \\ 0 \\ -5 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ -0.25 \\ 0 \\ -1 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ -0.75 \\ 0 \\ -6 \\ 0 \end{bmatrix}
\end{aligned}$$

Rotations	Rigid-Body Motions
$R \in SO(3) : 3 \times 3$ matrices $R^T R = I, \det R = 1$	$T \in SE(3) : 4 \times 4$ matrices $T = \begin{bmatrix} R & p \\ 0 & 1 \end{bmatrix},$ where $R \in SO(3), p \in \mathbb{R}^3$
$R^{-1} = R^T$	$T^{-1} = \begin{bmatrix} R^T & -R^T p \\ 0 & 1 \end{bmatrix}$
change of coordinate frame: $R_{ab}R_{bc} = R_{ac}, R_{ab}p_b = p_a$	change of coordinate frame: $T_{ab}T_{bc} = T_{ac}, T_{ab}p_b = p_a$
rotating a frame {b}: $R = \text{Rot}(\hat{\omega}, \theta)$ $R_{sb'} = RR_{sb}$: rotate θ about $\hat{\omega}_s = \hat{\omega}$ $R_{sb''} = R_{sb}R$: rotate θ about $\hat{\omega}_b = \hat{\omega}$	displacing a frame {b}: $T = \begin{bmatrix} \text{Rot}(\hat{\omega}, \theta) & p \\ 0 & 1 \end{bmatrix}$ $T_{sb'} = TT_{sb}$: rotate θ about $\hat{\omega}_s = \hat{\omega}$ (moves {b} origin), translate p in {s} $T_{sb''} = T_{sb}T$: translate p in {b}, rotate θ about $\hat{\omega}$ in new body frame
unit rotation axis is $\hat{\omega} \in \mathbb{R}^3$, where $\ \hat{\omega}\ = 1$	“unit” screw axis is $\mathcal{S} = \begin{bmatrix} \omega \\ v \end{bmatrix} \in \mathbb{R}^6$, where either (i) $\ \omega\ = 1$ or (ii) $\omega = 0$ and $\ v\ = 1$
	for a screw axis $\{q, \hat{s}, h\}$ with finite h , $\mathcal{S} = \begin{bmatrix} \omega \\ v \end{bmatrix} = \begin{bmatrix} \hat{s} \\ -\hat{s} \times q + h\hat{s} \end{bmatrix}$
angular velocity is $\omega = \hat{\omega}\dot{\theta}$	twist is $\mathcal{V} = \mathcal{S}\dot{\theta}$

Rotations (cont.)	Rigid-Body Motions (cont.)
for any 3-vector, e.g., $\omega \in \mathbb{R}^3$,	for $\mathcal{V} = \begin{bmatrix} \omega \\ v \end{bmatrix} \in \mathbb{R}^6$,
$[\omega] = \begin{bmatrix} 0 & -\omega_3 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ -\omega_2 & \omega_1 & 0 \end{bmatrix} \in so(3)$	$[\mathcal{V}] = \begin{bmatrix} [\omega] & v \\ 0 & 0 \end{bmatrix} \in se(3)$
identities, $\omega, x \in \mathbb{R}^3, R \in SO(3)$: $[\omega] = -[\omega]^T, [\omega]x = -[x]\omega,$ $[\omega][x] = ([x][\omega])^T, R[\omega]R^T = [R\omega]$	(the pair (ω, v) can be a twist \mathcal{V} or a “unit” screw axis \mathcal{S} , depending on the context)
$\dot{R}R^{-1} = [\omega_s], R^{-1}\dot{R} = [\omega_b]$	$\dot{T}T^{-1} = [\mathcal{V}_s], T^{-1}\dot{T} = [\mathcal{V}_b]$
	$[Ad_T] = \begin{bmatrix} R & 0 \\ [p]R & R \end{bmatrix} \in \mathbb{R}^{6 \times 6}$
	identities: $[Ad_T]^{-1} = [Ad_{T^{-1}}],$ $[Ad_{T_1}][Ad_{T_2}] = [Ad_{T_1 T_2}]$
change of coordinate frame: $\hat{\omega}_a = R_{ab}\hat{\omega}_b, \omega_a = R_{ab}\omega_b$	change of coordinate frame: $\mathcal{S}_a = [Ad_{T_{ab}}]\mathcal{S}_b, \mathcal{V}_a = [Ad_{T_{ab}}]\mathcal{V}_b$
exp coords for $R \in SO(3)$: $\hat{\omega}\theta \in \mathbb{R}^3$	exp coords for $T \in SE(3)$: $\mathcal{S}\theta \in \mathbb{R}^6$
exp : $[\hat{\omega}]\theta \in so(3) \rightarrow R \in SO(3)$ $R = \text{Rot}(\hat{\omega}, \theta) = e^{[\hat{\omega}]\theta} =$ $I + \sin \theta [\hat{\omega}] + (1 - \cos \theta) [\hat{\omega}]^2$	exp : $[\mathcal{S}]\theta \in se(3) \rightarrow T \in SE(3)$ $T = e^{[\mathcal{S}]\theta} = \begin{bmatrix} e^{[\omega]\theta} & * \\ 0 & 1 \end{bmatrix}$ where $*$ = $(I\theta + (1 - \cos \theta)[\omega] + (\theta - \sin \theta)[\omega]^2)v$
log : $R \in SO(3) \rightarrow [\hat{\omega}]\theta \in so(3)$ algorithm in Section 3.2.3.3	log : $T \in SE(3) \rightarrow [\mathcal{S}]\theta \in se(3)$ algorithm in Section 3.3.3.2
moment change of coord frame: $m_a = R_{ab}m_b$	wrench change of coord frame: $\mathcal{F}_a = (m_a, f_a) = [Ad_{T_{ba}}]^T \mathcal{F}_b$

5 Homework : Chapter 3

- Please solve and submit Exercise 3.5, 3.6, 3.7, 3.14, 3.17, 3.18, 3.21, 3.22, 3.23, 3.24, 3.25, 3.39, till April 9 (upload it as a pdf form or email me)
- If you let me know what the numbers you cannot solve until April 7, I will include the solving process in the next lecture.